

**Warsaw University  
of Technology**



**Faculty of Power and  
Aeronautical Engineering**

WARSAW UNIVERSITY OF TECHNOLOGY

Institute of Aeronautics and Applied Mechanics

# Finite element method (FEM)

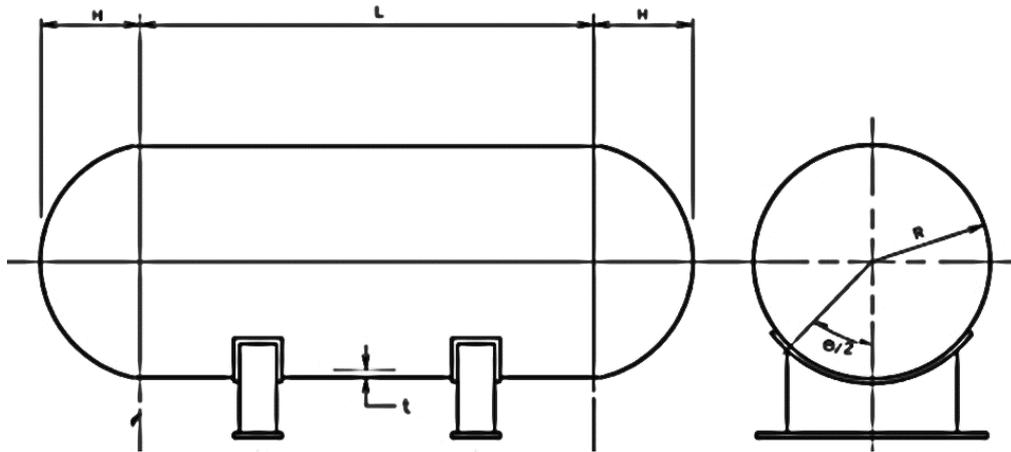
3D shell finite element

06.2021

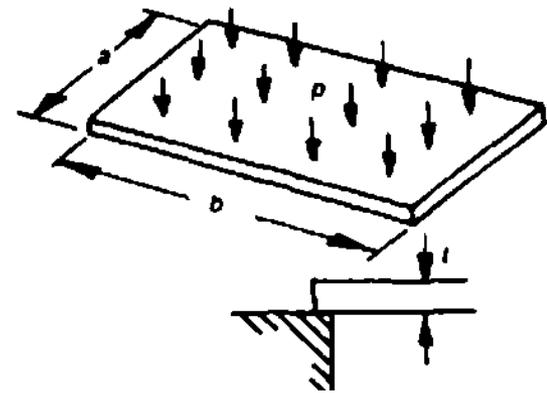
## Shells and plates

Thin-shells and plates models can be applied to analyze the following constructions:

- aircraft fuselage, the wing cover,
- boat hull,
- roof (floor) of building.



Thin shell of revolution



Rectangular plate

## Examples of plates and shells



aircraft skins (shell model)



construction of a building (plate model)

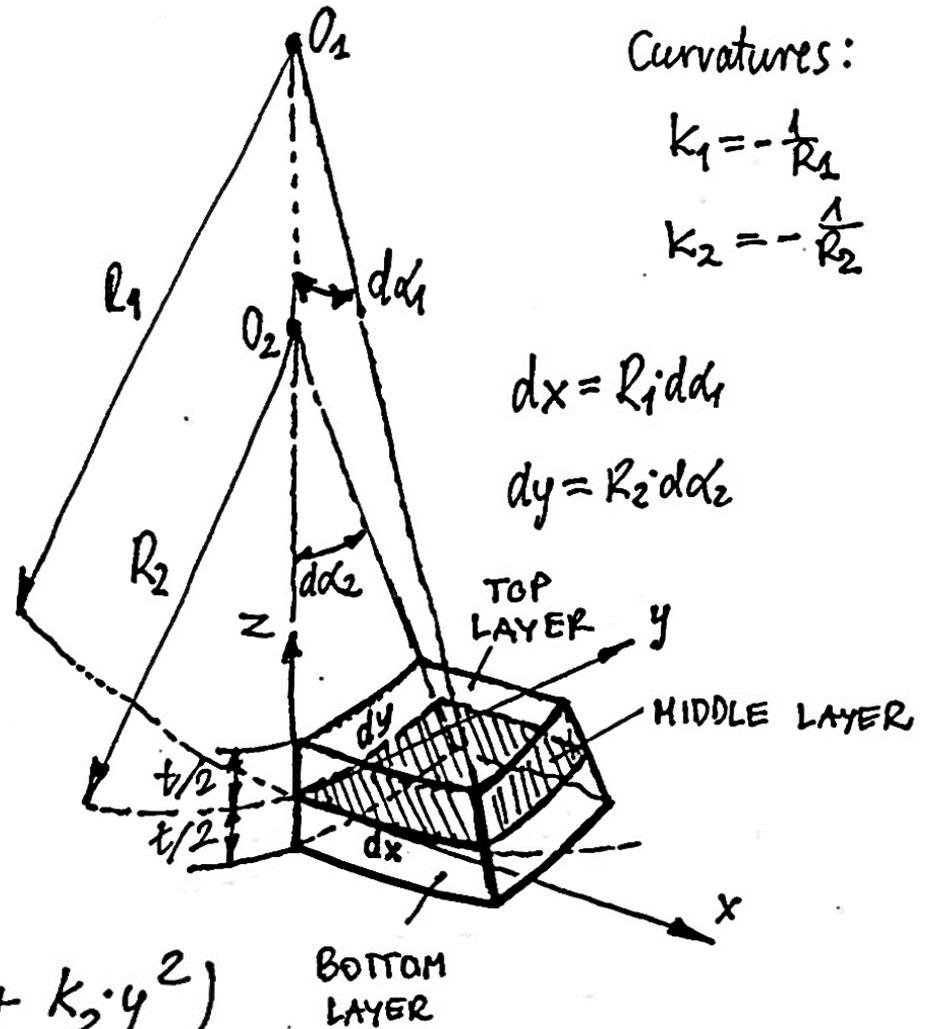


a motor yacht:  
the hull (shell), the deck (plate)

# LINEAR THEORY OF THIN SHELLS

types of shells:

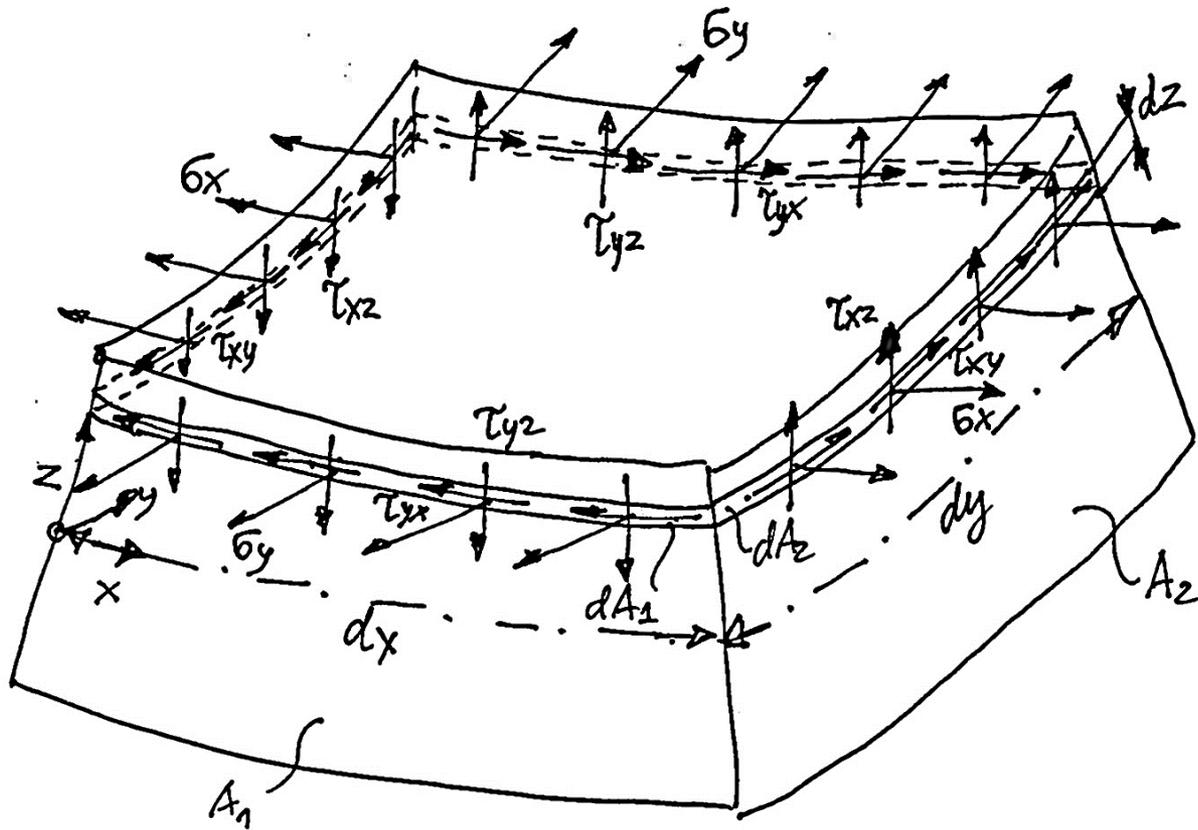
- elliptical
- cylindrical
- spherical
- toroidal
- hyperbolic



$$z = 0.5 (K_1 x^2 + 2K_{12} xy + K_2 y^2)$$

Internal force at level  $z$  in a small area  $dA_2$ :

$$\begin{aligned} \sigma_x \cdot dz \cdot (R_2 - z) d\alpha_2 &= \sigma_x \cdot dz \left(1 - \frac{z}{R_2}\right) R_2 d\alpha_2 = \\ &= \sigma_x \left(1 - \frac{z}{R_2}\right) dz dy \end{aligned}$$



Internal force at level  $z$  in a small area  $dA_2$  per unit length:

$$dN_x = \frac{\sigma_x \left(1 - \frac{z}{R_2}\right) dz dy}{dy} = \sigma_x \left(1 - \frac{z}{R_2}\right) dz$$

Internal force on a crosssectional area  $A_2$  per unit length:

$$N_x = \int_{-\frac{t}{2}}^{\frac{t}{2}} \sigma_x \left(1 - \frac{z}{R_2}\right) dz \quad \left(\frac{N}{m}\right)$$

$$\frac{z}{R_1} \approx 0, \quad \frac{z}{R_2} = 0 \Rightarrow$$

## Internal forces

$$n_x = \int_{-\frac{t}{2}}^{\frac{t}{2}} \sigma_x dz, \quad n_y = \int_{-\frac{t}{2}}^{\frac{t}{2}} \sigma_y dz, \quad n_{xy} = n_{yx} = \int_{-\frac{t}{2}}^{\frac{t}{2}} \tau_{xy} dz$$

$$m_x = \int_{-\frac{t}{2}}^{\frac{t}{2}} \sigma_x \cdot z dz, \quad m_y = \int_{-\frac{t}{2}}^{\frac{t}{2}} \sigma_y \cdot z dz, \quad m_{xy} = m_{yx} = \int_{-\frac{t}{2}}^{\frac{t}{2}} \tau_{xy} \cdot z dz \quad \left(\frac{Nm}{m}\right)$$

$$t_x = \frac{\partial m_x}{\partial x} + \frac{\partial m_{xy}}{\partial y}, \quad t_y = \frac{\partial m_y}{\partial y} + \frac{\partial m_{xy}}{\partial x}$$

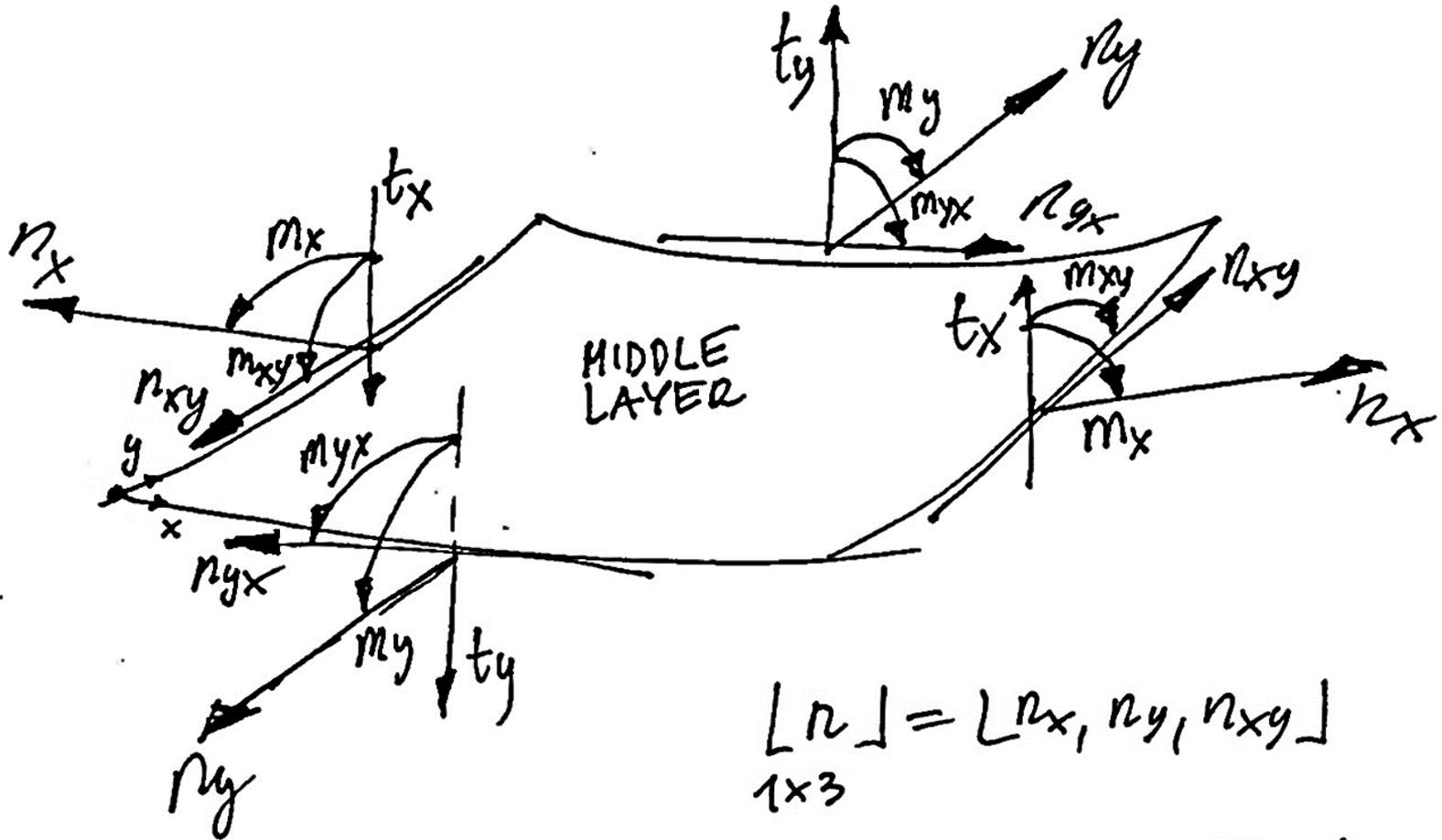
$n$  - normal force per unit length

$t$  - shear force per unit length

$m_x, m_y$  - bending moments per unit length

$m_{xy}$  - torque per unit length

# Internal forces



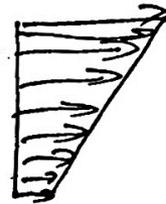
$$[n] = [n_x, n_y, n_{xy}]$$

1x3

$$[m] = [m_x, m_y, m_{xy}]$$

1x3

# Stress components



TOTAL STRESS

(=)

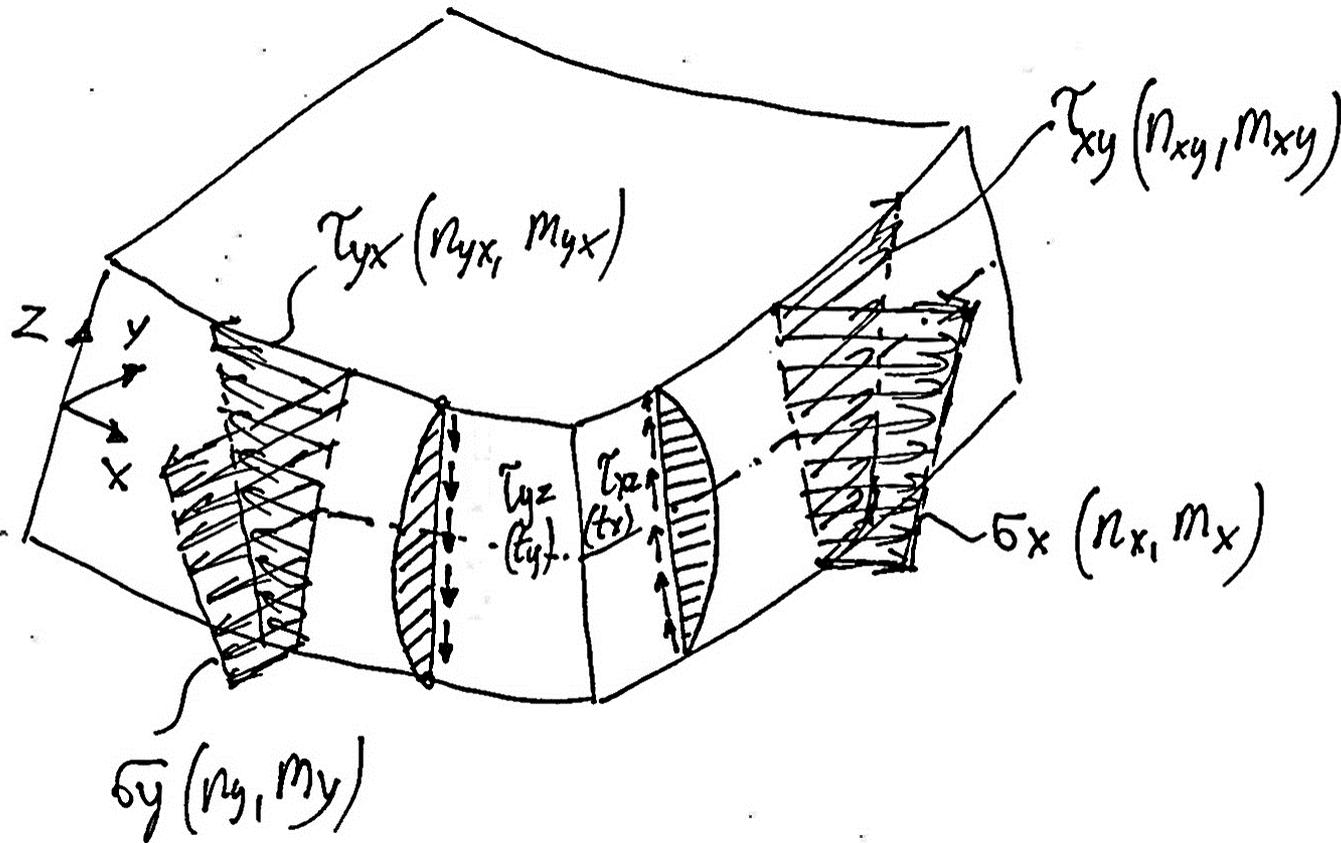


MEMBRANE STRESS

(+)



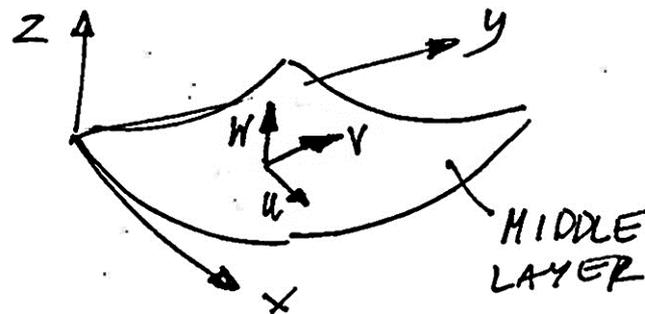
BENDING STRESS



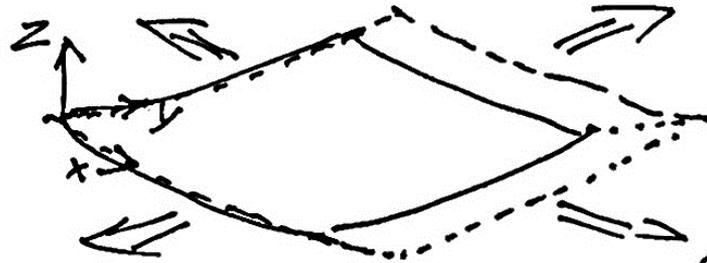
# MEMBRANE STRAIN:

1°) deformation of a middle layer in plane  $xy$

2°) deformation of a middle layer along  $z$  axis



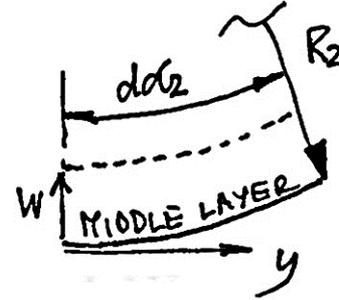
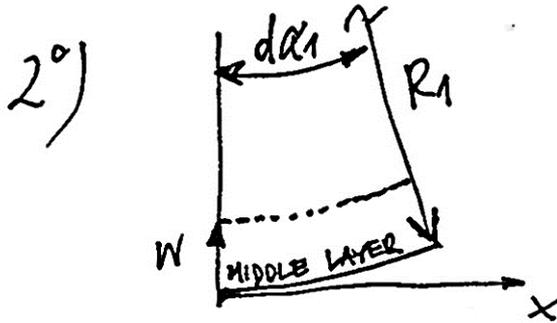
1°)



$$\epsilon_y^{1°} = \frac{\partial v}{\partial y}$$

$$\epsilon_x^{1°} = \frac{\partial u}{\partial x}$$

# MEMBRANE STRAIN



$$\epsilon_x^{2^\circ} = \frac{(R_1 - W) da_1 - R_1 da_1}{R_1 da_1} = -\frac{W}{R_1} = k_1 \cdot W$$

$$\epsilon_y^{2^\circ} = \frac{(R_2 - W) da_2 - R_2 da_2}{R_2 da_2} = -\frac{W}{R_2} = k_2 \cdot W$$

## MEMBRANE STRAIN

10) + 2°)

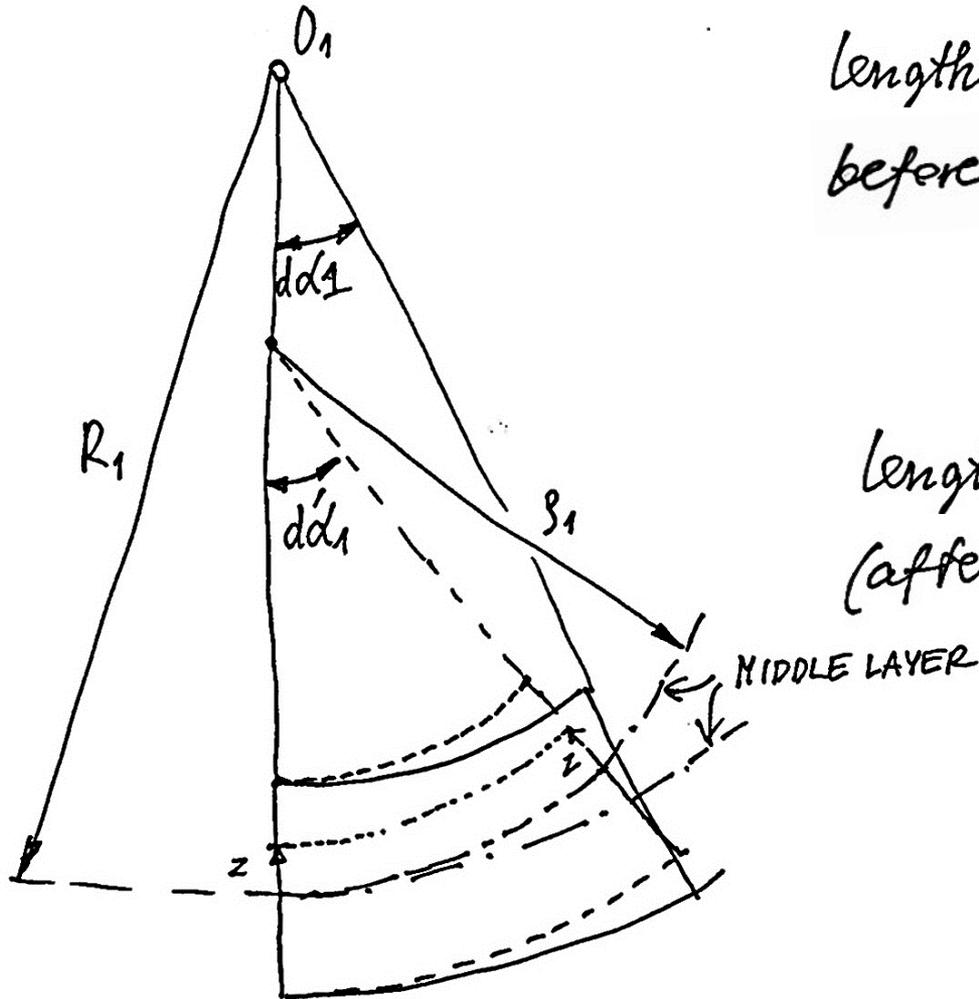
$$\epsilon_x^{\text{MID}} = \frac{\partial u}{\partial x} + k_1 \cdot w$$

$$\epsilon_y^{\text{MID}} = \frac{\partial v}{\partial y} + k_2 \cdot w$$

$$\gamma_{xy}^{\text{MID}} = \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} + 2k_2 \cdot w$$

## BENDING STRAIN :

3°) deformation of the layer at level  $z$



length of the layer at level  $z$   
before deformation:

$$R_1 \left(1 - \frac{z}{R_1}\right) \cdot d\alpha_1$$

length of the middle layer  
(after and before deformation):

$$R_1 \cdot d\alpha_1 = \beta_1 \cdot dd'_1$$

$$\varepsilon_x(z) = \frac{S_1 \left(1 - \frac{z}{S_1}\right) d\alpha_1' - R_1 \left(1 - \frac{z}{R_1}\right) d\alpha_1}{R_1 \left(1 - \frac{z}{R_1}\right) d\alpha_1} =$$

$$= \frac{S_1 \left(1 - \frac{z}{S_1}\right) \frac{R_1}{S_1} d\alpha_1 - R_1 \left(1 - \frac{z}{R_1}\right) d\alpha_1}{R_1 \left(1 - \frac{z}{R_1}\right) d\alpha_1} =$$

$$= \frac{\left(1 - \frac{z}{S_1}\right) - \left(1 - \frac{z}{R_1}\right)}{\left(1 - \frac{z}{R_1}\right)} = \frac{1 - \frac{z}{S_1}}{\underbrace{1 - \frac{z}{R_1}}_{\approx 1}} - 1 = -\frac{z}{S_1} =$$

$$= -\frac{\partial^2 W}{\partial x^2} \cdot z = \kappa_x \cdot z$$

$$\varepsilon_y(z) = -\frac{\partial^2 W}{\partial y^2} \cdot z = \kappa_y \cdot z \quad ; \quad \gamma_{xy}(z) = -2 \frac{\partial^2 W}{\partial x \partial y} \cdot z = \kappa_{xy} \cdot z$$

TOTAL (MEMBRANE + BENDING) STRAIN VECTOR

$$\begin{Bmatrix} \epsilon_x \\ \epsilon_y \\ \gamma_{xy} \end{Bmatrix} = \begin{Bmatrix} \epsilon_x^{MID} \\ \epsilon_y^{MID} \\ \gamma_{xy}^{MID} \end{Bmatrix} + \begin{Bmatrix} \epsilon_x(z) \\ \epsilon_y(z) \\ \gamma_{xy}(z) \end{Bmatrix} = \underbrace{\begin{Bmatrix} \epsilon_x^{MID} \\ \epsilon_y^{MID} \\ \gamma_{xy}^{MID} \end{Bmatrix}}_{3 \times 1} + \underbrace{\begin{Bmatrix} \kappa_x \\ \kappa_y \\ \kappa_{xy} \end{Bmatrix}}_{3 \times 1} \cdot z$$

$$\underbrace{\{\epsilon\}}_{3 \times 1} = \underbrace{\{\epsilon^{MID}\}}_{3 \times 1} + \underbrace{\{\kappa\}}_{3 \times 1} \cdot z$$

## STRESS COMPONENTS

ASSUMING PLANE STRESS CONDITION

$$[D] = \frac{E}{1-\nu^2} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & \frac{1}{2}(1-\nu) \end{bmatrix}$$

$$\begin{matrix} \{ \sigma \} \\ 3 \times 1 \end{matrix} = \begin{matrix} \left\{ \begin{matrix} \sigma_x \\ \sigma_y \\ \tau_{xy} \end{matrix} \right\} \\ 3 \times 1 \end{matrix} = \begin{matrix} [D] \\ 3 \times 3 \end{matrix} \cdot \begin{matrix} \{ \epsilon \} \\ 3 \times 1 \end{matrix} = \begin{matrix} [D] \\ 3 \times 3 \end{matrix} \cdot \begin{matrix} \{ \epsilon^{HID} \} \\ 3 \times 1 \end{matrix} + \begin{matrix} [D] \\ 3 \times 3 \end{matrix} \cdot \begin{matrix} \{ \alpha \epsilon \} \\ 3 \times 1 \end{matrix} \cdot z$$

# INTERNAL FORCES

$$\left\{ n \right\}_{3 \times 1} = \left\{ \begin{matrix} n_x \\ n_y \\ n_{xy} \end{matrix} \right\} = \int_{-\frac{t}{2}}^{\frac{t}{2}} \left\{ \begin{matrix} \sigma_x \\ \sigma_y \\ \tau_{xy} \end{matrix} \right\} dz = \underbrace{t [D]}_{[D_n]_{3 \times 3}} \cdot \left\{ \epsilon^{HID} \right\}_{3 \times 1} + \left\{ \begin{matrix} 0 \\ 0 \\ 0 \end{matrix} \right\} = [D_n]_{3 \times 3} \cdot \left\{ \epsilon^{HID} \right\}_{3 \times 1}$$

$$\left\{ m \right\}_{3 \times 1} = \left\{ \begin{matrix} m_x \\ m_y \\ m_{xy} \end{matrix} \right\} = \int_{-\frac{t}{2}}^{\frac{t}{2}} \left\{ \begin{matrix} \sigma_x \\ \sigma_y \\ \tau_{xy} \end{matrix} \right\} z dz = \left\{ \begin{matrix} 0 \\ 0 \\ 0 \end{matrix} \right\} + \frac{t^3}{12} [D]_{3 \times 3} \cdot \left\{ \kappa \right\} =$$

$$= \frac{Et^3}{12(1-\nu^2)} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & \frac{1}{2}(1-\nu) \end{bmatrix} \cdot \left\{ \begin{matrix} \kappa_x \\ \kappa_y \\ \kappa_{xy} \end{matrix} \right\} = [D_m]_{3 \times 3} \cdot \left\{ \kappa \right\}_{3 \times 1}$$

## STRESS COMPONENTS AS FUNCTIONS OF INTERNAL FORCES

$$\begin{array}{c}
 \{\sigma\} = [D] \{\epsilon^{MID}\} + [D] \cdot \{d\} \cdot z = \frac{1}{t} \cdot \{n\} + \frac{12}{t^3} \{m\} \cdot z \\
 \begin{array}{ccc}
 3 \times 1 & 3 \times 3 & 3 \times 1 \\
 & & 3 \times 3 \\
 & & 3 \times 1
 \end{array} \\
 \parallel \\
 \frac{1}{t} \cdot [D]^{-1} \cdot \{n\} \qquad \frac{12}{t^3} [D]^{-1} \cdot \{m\} \\
 \begin{array}{ccc}
 3 \times 3 & 3 \times 1 & 3 \times 3 \quad 3 \times 2
 \end{array}
 \end{array}$$

normal stresses:

$$\sigma_x = \frac{n_x}{t} + \frac{12 m_x}{t^3} \cdot z$$

$$\sigma_y = \frac{n_y}{t} + \frac{12 m_y}{t^3} \cdot z$$

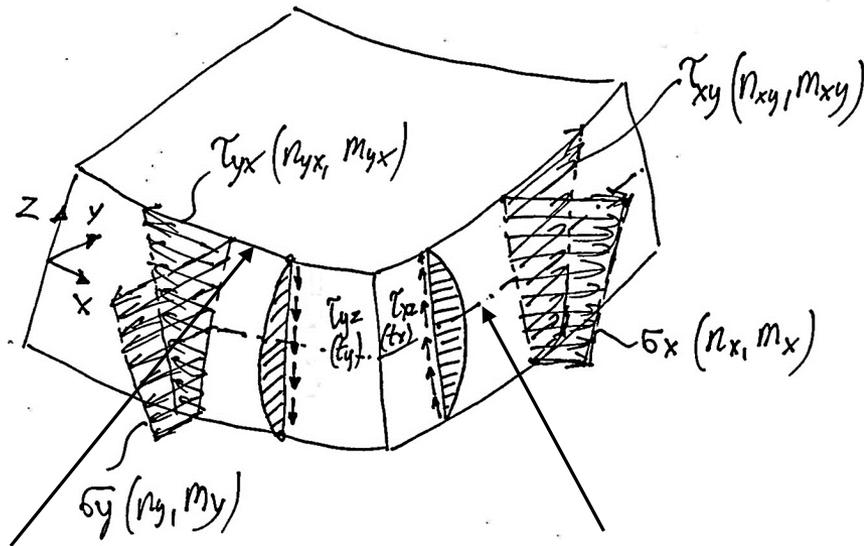
shear stresses:

$$\tau_{xy} = \tau_{yx} = \frac{n_{xy}}{t} + \frac{12 m_{xy}}{t^3} \cdot z$$

$$\tau_{xz} = \frac{3 t_x}{2t} \left( 1 - \frac{4z^2}{t^2} \right)$$

$$\tau_{yz} = \frac{3 t_y}{2t} \left( 1 - \frac{4z^2}{t^2} \right)$$

# MAXIMUM VALUES OF STRESS COMPONENTS



TOP LAYER

$$\sigma_x^{\text{TOP}} = \frac{n_x}{t} + \frac{6m_x}{t^2}$$

$$\sigma_y^{\text{TOP}} = \frac{n_y}{t} + \frac{6m_y}{t^2}$$

$$\tau_{xy}^{\text{TOP}} = \frac{n_{xy}}{t} + \frac{6m_{xy}}{t^2}$$

$$\tau_{xz}^{\text{TOP}} = 0$$

$$\tau_{yz}^{\text{TOP}} = 0$$

MIDDLE LAYER

$$\sigma_x^{\text{MID}} = \frac{n_x}{t}$$

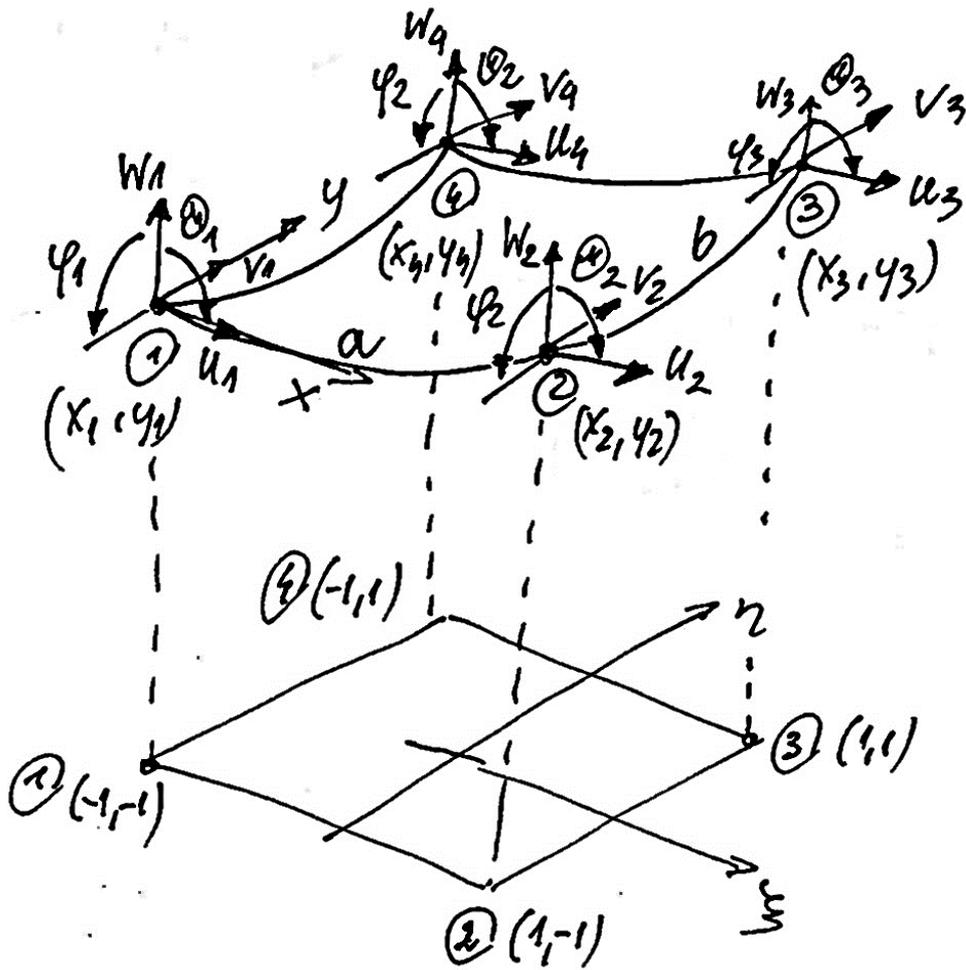
$$\sigma_y^{\text{MID}} = \frac{n_y}{t}$$

$$\tau_{xy}^{\text{MID}} = \frac{n_{xy}}{t}$$

$$\tau_{xz}^{\text{MID}} = \frac{3}{2} \cdot \frac{t_x}{t}$$

$$\tau_{yz}^{\text{MID}} = \frac{3}{2} \cdot \frac{t_y}{t}$$

# AN ISOPARAMETRIC SHELL FINITE ELEMENT



$$n = 4$$

$$n_0 = 5 \Rightarrow$$

$$n_e = 4 \cdot 5 = 20$$

$$y_i = \frac{dw}{dy} \Big|_i$$

$$\theta_i = - \frac{\partial w}{\partial x} \Big|_i$$

Local vector of nodal parameters (three parts)

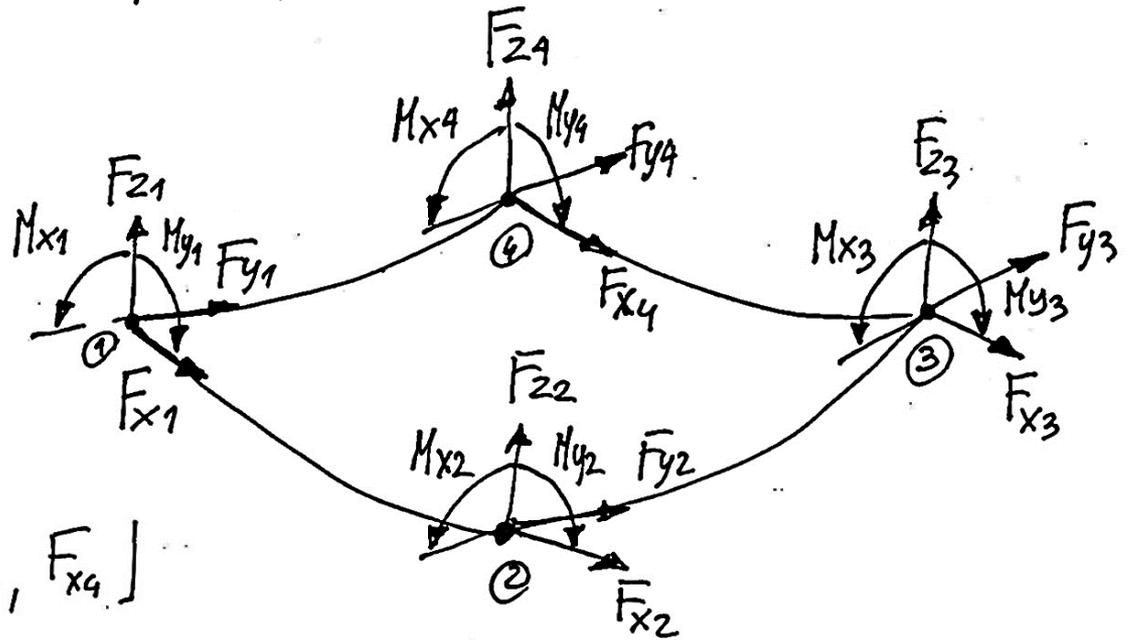
$$Lq_u|_e = [u_1, u_2, u_3, u_4]_e$$

$$Lq_v|_e = [v_1, v_2, v_3, v_4]_e$$

$$Lq_w|_e = [w_1, \psi_1, \theta_1, w_2, \psi_2, \theta_2, w_3, \psi_3, \theta_3, w_4, \psi_4, \theta_4]_e$$

$$Lq|_e = [Lq_u|_e, Lq_v|_e, Lq_w|_e]_e$$

# Local load vector (three parts)



$$[F_x]_e = [F_{x1}, F_{x2}, F_{x3}, F_{x4}]$$

$1 \times 4$

$$[F_y]_e = [F_{y1}, F_{y2}, F_{y3}, F_{y4}]$$

$1 \times 4$

$$[F_z]_e = [F_{z1}, M_{x1}, M_{y1}, F_{z2}, M_{x2}, M_{y2}, F_{z3}, M_{x3}, M_{y3}, F_{z4}, M_{x4}, M_{y4}]$$

$1 \times 12$

$$[F]_e = [[F_x]_e, [F_y]_e, [F_z]_e]_e$$

$1 \times 20$

## Nodal approximation and shape functions

$$u = N_1 \cdot u_1 + N_2 \cdot u_2 + N_3 \cdot u_3 + N_4 \cdot u_4$$

$$v = N_1 \cdot v_1 + N_2 \cdot v_2 + N_3 \cdot v_3 + N_4 \cdot v_4$$

$$w = N_{11} \cdot w_1 + N_{12} \cdot \psi_1 + N_{13} \cdot \phi_1 + N_{21} w_2 + N_{22} \psi_2 + N_{23} \phi_2 + \\ + N_{31} w_3 + N_{32} \psi_3 + N_{33} \phi_3 + N_{41} w_4 + N_{42} \psi_4 + N_{43} \phi_4$$

$$\underset{1 \times 4}{[N]} = [N_1, N_2, N_3, N_4] \quad (\text{polynomials of } \xi \text{ and } \eta)$$

$$\underset{1 \times 12}{[N_w]} = [N_{11}, N_{12}, N_{13}, N_{21}, N_{22}, N_{23}, N_{31}, N_{32}, N_{33}, N_{41}, N_{42}, N_{43}] \\ (\text{Hermite polynomials})$$

# Nodal approximation and shape functions

$$u = \underset{1 \times 4}{[N]} \cdot \{q_u\}_e$$

$$v = \underset{1 \times 4}{[N]} \cdot \{q_v\}_e$$

$$w = \underset{1 \times 12}{[N_w]} \cdot \{q_w\}_e$$

$$\underset{3 \times 1}{\{u\}} = \underset{3 \times 1}{\begin{Bmatrix} u \\ v \\ w \end{Bmatrix}} = \underset{3 \times 20}{\begin{bmatrix} \underset{1 \times 4}{[N]} & \underset{1 \times 4}{[0]} & \underset{1 \times 12}{[0]} \\ \underset{1 \times 4}{[0]} & \underset{1 \times 4}{[N]} & \underset{1 \times 12}{[0]} \\ \underset{1 \times 4}{[0]} & \underset{1 \times 4}{[0]} & \underset{1 \times 12}{[N_w]} \end{bmatrix}} \cdot \underset{20 \times 1}{\{q\}_e} = \underset{3 \times 20}{[N]} \cdot \underset{20 \times 1}{\{q\}_e}$$

MEMBRANE STRAIN:

$$\epsilon_x^{MID} = \frac{\partial u}{\partial x} + k_1 \cdot W = \frac{\partial \underline{[N]}_{1 \times 4}}{\partial x} \cdot \{q_u\}_e + k_1 \cdot \underline{[N_w]}_{1 \times 12} \cdot \{q_w\}_e$$

$$\epsilon_y^{MID} = \frac{\partial v}{\partial y} + k_2 \cdot W = \frac{\partial \underline{[N]}_{1 \times 4}}{\partial y} \{q_v\}_e + k_2 \cdot \underline{[N_w]}_{1 \times 12} \cdot \{q_w\}_e$$

$$\gamma_{xy}^{MID} = \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} + k_{12} \cdot W = \frac{\partial \underline{[N]}_{1 \times 4}}{\partial y} \{q_u\}_e + \frac{\partial \underline{[N]}_{1 \times 4}}{\partial x} \{q_v\}_e + k_{12} \underline{[N_w]}_{1 \times 12} \cdot \{q_w\}_e$$

BENDING STRAIN (FUNCTION OF CURVATURES):

$$\kappa_x = -\frac{\partial^2 W}{\partial x^2} = -\frac{\partial^2 \left[ \frac{N_w}{1 \times 12} \right]}{\partial x^2} \cdot \left\{ q_w \right\}_e$$

$$\kappa_y = -\frac{\partial^2 W}{\partial y^2} = -\frac{\partial^2 \left[ \frac{N_w}{1 \times 12} \right]}{\partial y^2} \cdot \left\{ q_w \right\}_e$$

$$\kappa_{xy} = -2 \frac{\partial^2 W}{\partial x \partial y} = -\frac{\partial^2 \left[ \frac{N_w}{1 \times 12} \right]}{\partial x \partial y} \left\{ q_w \right\}_e$$

# STRAIN-DISPLACEMENT MATRIX

$$\begin{Bmatrix} \{ \epsilon \}^{mid} \\ \{ \epsilon \} \\ \{ \epsilon \} \end{Bmatrix}_{6 \times 1} = \begin{bmatrix} \frac{\partial}{\partial x} [N]_{1 \times 4} & [0]_{1 \times 4} & K_1 [N_w]_{1 \times 12} \\ [0]_{1 \times 4} & \frac{\partial}{\partial y} [N]_{1 \times 4} & K_2 [N_w]_{1 \times 12} \\ \frac{\partial}{\partial y} [N]_{1 \times 4} & \frac{\partial}{\partial x} [N]_{1 \times 4} & K_{12} [N_w]_{1 \times 12} \\ [0]_{3 \times 4} & [0]_{3 \times 4} & \begin{matrix} \frac{\partial}{\partial x} \left( \frac{\partial}{\partial x} [N_w]_{1 \times 12} \right) \\ \frac{\partial}{\partial y} \left( \frac{\partial}{\partial y} [N_w]_{1 \times 12} \right) \\ -2 \frac{\partial}{\partial y} \left( \frac{\partial}{\partial x} [N_w]_{1 \times 12} \right) \end{matrix} \end{bmatrix}_{6 \times 20} \cdot \begin{Bmatrix} \{ q_u \} \\ \{ q_v \} \\ \{ q_w \} \end{Bmatrix}_{20 \times 1}$$

# STRAIN-DISPLACEMENT MATRIX

$$\{q_{uv}\}_e = \begin{Bmatrix} \{q_u\}_e \\ \{q_v\}_e \end{Bmatrix}$$

$8 \times 1$        $4 \times 1$        $4 \times 1$

$$\begin{Bmatrix} \{E\}^{HID} \\ \{d\} \end{Bmatrix} = \begin{bmatrix} [B_M] & [B_S] \\ [O] & [B_B] \end{bmatrix} \cdot \begin{Bmatrix} \{q_{uv}\}_e \\ \{q_w\}_e \end{Bmatrix}$$

$3 \times 1$        $3 \times 1$        $3 \times 8$        $3 \times 12$        $8 \times 1$        $12 \times 1$

$6 \times 1$

↖  
 strain - displacement  
 matrix

# STRAIN-DISPLACEMENT MATRIX

$$[B_M] = \begin{bmatrix} \frac{\partial}{\partial x} [N]_{1 \times 4} & [0]_{1 \times 4} \\ [0]_{1 \times 4} & \frac{\partial}{\partial y} [N]_{1 \times 4} \\ \frac{\partial}{\partial y} [N]_{1 \times 4} & \frac{\partial}{\partial x} [N]_{1 \times 4} \end{bmatrix}$$

(middle layer)

$$[B_S] = \begin{bmatrix} k_1 [N_w]_{1 \times 12} \\ k_2 [N_w]_{1 \times 12} \\ k_{i2} [N_w]_{1 \times 12} \end{bmatrix}$$

(shell curvatures)

$$[B_B] = \begin{bmatrix} -\frac{\partial}{\partial x} \left( \frac{\partial}{\partial x} [N_w]_{1 \times 12} \right) \\ -\frac{\partial}{\partial y} \left( \frac{\partial}{\partial y} [N_w]_{1 \times 12} \right) \\ -2 \frac{\partial}{\partial y} \left( \frac{\partial}{\partial x} [N_w]_{1 \times 12} \right) \end{bmatrix}$$

(bending)

# ELASTIC STRAIN ENERGY

$$U_e = U_e(\{\epsilon\}_{3 \times 1}^{HID}) + U_e(\{\eta\}_{3 \times 1})$$

$$U_e(\{\epsilon\}_{3 \times 1}^{HID}) = \int_{A_e} \frac{1}{2} L E_{1 \times 3}^{HID} \cdot \{\eta\}_{3 \times 1} dA_e =$$

$$= \frac{1}{2} \int_{A_e} L q_{1 \times 20} \cdot \begin{bmatrix} [B_M]^T \\ [B_S]^T \end{bmatrix} \cdot [D_n]_{3 \times 3} \cdot \{\epsilon\}_{3 \times 1}^{HID} dA_e =$$

# ELASTIC STRAIN ENERGY

⇒

$$U_e(\{\epsilon\}^{H10}) = \frac{1}{2} L q \int_{A_e} \begin{bmatrix} [B_M]^T \\ 8 \times 3 \\ [B_S]^T \\ 12 \times 3 \end{bmatrix} [D_n] \cdot \begin{bmatrix} [B_M] \\ 3 \times 8 \\ [B_S] \\ 3 \times 12 \end{bmatrix} dA_e \cdot \{q\}_e =$$

$$= \frac{1}{2} L q \int_{A_e} \begin{bmatrix} [B_M]^T \\ 8 \times 3 \\ [B_S]^T \\ 12 \times 3 \end{bmatrix} \cdot \begin{bmatrix} [D_n] \cdot [B_M] \\ 3 \times 3 \quad 3 \times 8 \\ [D_n] \cdot [B_S] \\ 3 \times 3 \quad 3 \times 12 \end{bmatrix} dA_e \{q\}_e =$$

$$= \frac{1}{2} L q \int_{A_e} \begin{bmatrix} [B_M]^T [D_n] [B_M] & [B_M]^T [D_n] [B_S] \\ 8 \times 3 \quad 3 \times 3 \quad 3 \times 8 & 8 \times 3 \quad 3 \times 3 \quad 3 \times 12 \\ [B_S]^T [D_n] [B_M] & [B_S]^T [D_n] [B_S] \\ 12 \times 3 \quad 3 \times 3 \quad 3 \times 8 & 12 \times 3 \quad 3 \times 3 \quad 3 \times 12 \end{bmatrix} dA_e \{q\}_e$$

# ELASTIC STRAIN ENERGY

$$U_e(\{u\}) = \int_{Ae} \frac{1}{2} \cdot \underset{1 \times 3}{[k]} \cdot \underset{3 \times 1}{\{m\}} dA =$$

$$= \frac{1}{2} \int_{Ae} \underset{1 \times 20}{[q]} \cdot \begin{bmatrix} \underset{8 \times 3}{[0]} \\ \underset{12 \times 3}{[B_B]^T} \end{bmatrix} \cdot \underset{3 \times 3}{[D_m]} \cdot \underset{3 \times 1}{\{u\}} dA =$$

$$= \frac{1}{2} \underset{1 \times 20}{[q]} \int_{Ae} \begin{bmatrix} \underset{8 \times 3}{[0]} \\ \underset{12 \times 3}{[B_B]^T} \end{bmatrix} \underset{3 \times 3}{[D_m]} \cdot \begin{bmatrix} \underset{3 \times 8}{[0]} \\ \underset{3 \times 12}{[B_B]} \end{bmatrix} dAe \cdot \underset{20 \times 1}{\{q\}} =$$

# ELASTIC STRAIN ENERGY

$\Rightarrow$

$$U_e(\{q\}) = \frac{1}{2} L q_e \int_{A_e} \begin{bmatrix} [0] \\ 8 \times 3 \\ [B_B]^T \end{bmatrix} \cdot \begin{bmatrix} [D_m][0] \\ 3 \times 3 \quad 3 \times 8 \\ [D_m] \cdot [B_B] \\ 3 \times 3 \quad 3 \times 12 \end{bmatrix} dA_e \cdot \{q\}_e =$$

$$= \frac{1}{2} L q_e \int_{A_e} \begin{bmatrix} [0] & [0] \\ 8 \times 8 & 8 \times 12 \\ [0] & [B_B]^T [D_m] [B_B] \\ 12 \times 8 & 12 \times 3 \quad 3 \times 3 \quad 3 \times 12 \end{bmatrix} dA_e \{q\}_e \Rightarrow$$

## ELASTIC STRAIN ENERGY

$$\Rightarrow U_e = \underbrace{\frac{1}{2} L q_e}_{1 \times 20} \cdot \underbrace{[k]_e}_{20 \times 20} \cdot \underbrace{\{q\}_e}_{20 \times 1} \quad \text{where:}$$

$$[k]_e = \int_{A_e} \left[ \begin{array}{c|c} \begin{matrix} [B_M]^T [D_b] [B_M] \\ 8 \times 3 & 3 \times 3 & 3 \times 8 \end{matrix} & \begin{matrix} [B_M]^T [D_n] [B_S] \\ 8 \times 3 & 3 \times 3 & 3 \times 12 \end{matrix} \\ \hline \begin{matrix} [B_S]^T [D_n] [B_M] & [B_S]^T [D_n] [B_S] + [B_B]^T [D_m] [B_B] \\ 12 \times 3 & 3 \times 3 & 3 \times 8 & 12 \times 3 & 3 \times 3 & 3 \times 12 & 12 \times 3 & 3 \times 3 & 3 \times 12 \end{matrix} \end{array} \right] dA_e$$

Stiffness matrix of a shell finite element

## POTENTIAL ENERGY OF LOADING

$$W_e = \underset{1 \times 20}{L q \downarrow}_e \cdot \underset{20 \times 1}{\{F\}}_e$$